

**INSTRUCTION:**

This section consists of **FOUR (4)** essay questions. Answer **ALL** questions.

**ARAHAN:**

*Bahagian ini mengandungi EMPAT (4) soalan esei. Jawab SEMUA soalan.*

**QUESTION 1****SOALAN 1**

- CLO1  
C1 (a) State **FOUR (4)** comparisons between rotodynamics & rotary pumps.  
*Nyatakan EMPAT (4) perbandingan diantara pam rotodinamik dan rotari.*
- [4 marks]  
[4 markah]
- CLO1  
C2 (b) Identify **FIVE (5)** matters to take into consideration when purchasing a pump.  
*Nyatakan LIMA (5) perkara yang perlu diambil kira apabila membeli sebuah pam.*
- [10 marks]  
[10 markah]
- CLO2  
C3 (c) A single stage, single acting air compressor running at 11 rev/second, has a diameter of 0.40 m and stroke of 0.60 m. The clearance volume is 3% of the swept volume. The inlet pressure and temperature are 90 000 N/m<sup>2</sup> and 30°C respectively. The delivery pressure is 760 000 N/m<sup>2</sup>. Assume the index of polytropic is 1.3. Calculate:
- (i) Volume induced  
(ii) Indicated Power
- Sebuah pemampat udara salingan tindakan tunggal, satu peringkat mempunyai diameter 0.40 m dan lejang 0.60m. Pemampat beroperasi dengan kelajuan 11 putaran/saad. Isipadu kelegaan adalah 3% daripada isipadu tersapu. Tekanan masuk 85 000 N/m<sup>2</sup> dan suhu masuk 30 °C. Tekanan penghantaran 760 000 N/m<sup>2</sup>. Andaikan indeks politropik ialah 1.3.*
- Kira :*
- (i) Isipadu teraruh  
(ii) Kuasa tertunjuk
- [11 marks]  
[11 markah]

**QUESTION 2**  
**SOALAN 2**CLO1  
C2(a) Explain **FOUR (4)** types of liquid fuel.*Terangkan EMPAT (4) jenis bahan api cecair.*[4 marks]  
[4 markah]CLO2  
C2(b) Determine the stoichiometric mass air for 1 kg of Petrol  $C_2H_5O$ *Tentukan jisim udara stoikiometric untuk pembakaran 1 kg Petrol  $C_2H_5O$* [6 marks]  
[6 markah]CLO2  
C3

(c) The analysis of gas supply is as follow:

**85%  $CH_4$ , 3.5%  $C_2H_6$ , 0.6%  $C_3H_8$ , 0.4%  $C_4H_{10}$ , 10%  $N_2$ , and 0.5%  $CO_2$** 

Calculate:

- (i) The stoichiometric A/F ratio using the chemical equilibrium method.
- (ii) The A/F ratio and the dry and wet analysis of the products of combustion by volume, when 35% excess air is supplied.

*Analisis bekalan gas adalah seperti berikut:***85%  $CH_4$ , 3.5%  $C_2H_6$ , 0.6%  $C_3H_8$ , 0.4%  $C_4H_{10}$ , 10%  $N_2$ , and 0.5%  $CO_2$** *Kira:*

- (i) *Nisbah U/B stoikiometri menggunakan kaedah keseimbangan kimia.*
- (ii) *Nisbah U/B sebenar dan analisa basah dan kering hasil pembakaran mengikut isipadu, bila udara lebihan sebanyak 35% dibekalkan.*

[15 marks]  
[15 markah]

**QUESTION 3****SOALAN 3**CLO1  
C2

- (a) List **FOUR (4)** nozzle applications in engineering sector.

*Senaraikan EMPAT (4) aplikasi muncung dalam sektor kejuruteraan*

[4 marks]

[4 markah]

CLO2  
C3

- (c) The inlet condition of air to a convergent divergent nozzle is 2.2 MPa and 260°C. The exit pressure is 0.4 MPa with  $\gamma = 1.4$ . Throat area of 32.2 cm<sup>2</sup>, determine:

- (i) Critical Pressure
- (ii) Critical Temperature
- (iii) Critical Velocity
- (iv) The mass flowrate
- (v) The exit Temperature
- (vi) The exit Velocity
- (vii) The exit area.

*Keadaan salur masuk udara untuk muncung tumpu campa ialah 2.2 MPa dan 260 °C. Tekanan keluar ialah 0.4 MPa dengan nilai  $\gamma = 1.4$ . luas keratan rentas kerongkong ialah 32.2 cm<sup>2</sup>, kirakan:*

- (i) *Tekanan Kritikal*
- (ii) *Suhu Kritikal*
- (iii) *Halaju Kritikal*
- (iv) *Kadar alir jisim*
- (v) *Suhu pada keluaran*
- (vi) *Halaju pada keluaran*
- (vii) *Luas keratan rentas keluaran.*

[21 marks]

[21 markah]

## QUESTION 4

## SOALAN 4

- CLO1  
C1
- (a) List **FOUR (4)** desirable properties of a good refrigerant. (4 marks)  
*Berikan EMPAT (4) sifat-sifat bahan pendingin yang baik. (4 markah)*
- CLO2  
C2
- (b) Refrigerant 134a is the working fluid in a vapour-compression refrigeration cycle and operates on an ideal vapour compression refrigeration cycle between  $-15^{\circ}\text{C}$  and  $40^{\circ}\text{C}$ . Dry saturated vapour is delivered to the compressor where it is compressed isentropically. The condition of the liquid after condensation process is undercooled by  $5^{\circ}\text{C}$ . Illustrate the process cycle the temperature versus entropy (T-s) in a diagram. If the mass flow rate of refrigerant is  $0.085\text{ kg/s}$ , calculate the following:-
- Compressor power in kilowatt (kW)
  - Refrigerating effect in kilowatt (kW)
  - Coefficient of performance of refrigerator (C.O.P<sub>REF</sub>)
  - Coefficient of performance of heat pump, (C.O.P<sub>HP</sub>)
- [13 marks]  
[13 markah]
- CLO2  
C4
- (c) Atmospheric air with barometric pressure of  $1.013\text{ bar}$  has  $40^{\circ}\text{C}$  dry bulb temperature and  $26^{\circ}\text{C}$  wet bulb temperature. Without the aid of psychrometric chart, determine:
- Relative humidity
  - Saturation percentage
- Udara atmosfera dengan tekanan barometer  $1.013\text{ bar}$  mempunyai  $40^{\circ}\text{C}$  suhu bebuli kering dan  $26^{\circ}\text{C}$  suhu bebuli basah. Tanpa bantuan carta psikrometer, tentukan:*
- Kelembapan relatif*
  - Peratus ketepuan*
- [8 marks]  
[8 markah]

SOALAN TAMAT

**RUMUS DJL40032-POWER PLANT ENGINEERING 2  
NOZZEL**

$$C_2 = \sqrt{2(h_1 - h_2)} = \sqrt{2 \times Cp \times (T_1 - T_2)}$$

$$C_2 = 44.72 \sqrt{(h_1 - h_2)}$$

$$\frac{T_1}{T_2} = \left[ \frac{P_1}{P_2} \right]^{\frac{\gamma-1}{\gamma}}$$

$$\text{Mass flow, } \dot{m} = \frac{CA}{v}$$

$$\frac{P_c}{P_1} = \left( \frac{2}{\gamma+1} \right)^{\frac{\gamma}{\gamma-1}}$$

$$\frac{T_c}{T_1} = \frac{2}{\gamma+1}$$

$$Cc = \sqrt{(\gamma RT_c)}$$

**REFRIGERATION**

$$\text{WORK INPUT} = (h_2 - h_1)$$

$$\text{REFRIGERATION EFFECT} = (h_1 - h_4)$$

$$\text{COP}_{\text{ref}} = \frac{Q}{W} = \frac{(h_1 - h_4)}{(h_2 - h_1)}$$

**COMBUSTION**

**Stoichiometric Air-Fuel Ratio:**

$$\frac{100}{23.3} \left[ \frac{8}{3} C + 8H + S - O \right]$$

$$\text{Actual A/F Ratio: } \frac{100}{23.3} \left[ \frac{8}{3} C + 8H + S - O \right] + \frac{\text{excess.air}}{100} \times \frac{100}{23.3} \left[ \frac{8}{3} C + 8H + S - O \right]$$

## RUMUS DJL40032-POWER PLANT ENGINEERING 2

### PUMP

$$\text{NPSH} = \left[ \frac{P_a - P_v}{\rho g} \right] - h_f - h_v - h_{\text{fiting}} \pm h_s$$

$$\text{Velocity Head} = \frac{V^2}{2g}$$

$$\text{Pressure} = \rho g H_T \text{ N/m}^2$$

$$\text{Power} = \frac{\rho g Q H_T}{\eta_{\text{mech.pump}}} \text{ watt}$$

### AIR CONDITIONING

$$(\phi) = P_s / P_g$$

$$\psi = \frac{100\omega(P_B - P_g)}{0.622P_g}$$

$$\psi = \frac{p_s}{p_g} \left[ \frac{p_b - p_g}{p_b - p_s} \right]$$

$$\omega = 0.622 \times \left( \frac{P_s}{P_B - P_s} \right)$$

### COMPRESSOR

$$\text{Work input per cycle: } \frac{n}{n-1} \times m R (T_2 - T_1)$$

$$\text{Indicated Power: } \frac{n}{n-1} \times pV \left\{ \left( \frac{P_2}{P_1} \right)^{\frac{n-1}{n}} - 1 \right\} \times N$$

$$\text{Compressor mechanical efficiency} = \frac{\text{Indicated Power}}{\text{Shaft Power}}$$

$$\text{Delivery temperature } T_2 = T_1 \times \left[ \frac{P_2}{P_1} \right]^{\frac{n-1}{n}}$$

$$\text{Volumetric efficiency} = 1 - \frac{V_c}{V_s} \times \left\{ \left( \frac{P_2}{P_1} \right)^{\frac{1}{n}} - 1 \right\}$$

**TABLE FOR WET AND DRY ANALYSIS (CHEMICAL METHOD)**

<b>PRODUCT</b>	<b>KMOL/KMOL FUEL</b>	<b>% BY VOL. (WET)</b>	<b>% BY VOL. (DRY)</b>
<b>TOTAL (100%)</b>		<b>100</b>	<b>100</b>
	<b>WET =</b>		
	<b>DRY =</b>		

**TABLE FOR WET AND DRY ANALYSIS**

COMBUSTION RESULT	MASS/KG FUEL	% MASS	M	MASS/KG FUEL / M	%WET ANALYSIS	%DRY ANALYSIS
<b>TOTAL</b>			<b>WET</b>			
			<b>DRY</b>			





R134a - TetraFlouroEthane Saturation Properties - Temperature Table (-40°C - 20°C)

Temp °C	Pressure kPa	volume (m <sup>3</sup> /kg)		vg	vf	enthalpy (kJ/kg)				entropy (kJ/kg.K)		
		vf	vg			hf	hfg	hg	sf	sfg	sg	
-40	51.2	0.0007054	0.3611	0.00	225.86	0.0000	0.9687	0.9687				
-36	62.9	0.0007112	0.2977	5.04	223.35	0.0214	0.9418	0.9632				
-32	76.7	0.0007172	0.2473	10.10	220.81	0.0425	0.9157	0.9582				
-28	92.7	0.0007234	0.2068	15.20	218.23	0.0634	0.8902	0.9536				
-26	101.7	0.0007265	0.1896	17.76	216.92	0.0738	0.8777	0.9515				
-24	111.3	0.0007297	0.1741	20.33	215.60	0.0841	0.8653	0.9495				
-22	121.7	0.0007329	0.1601	22.91	214.26	0.0944	0.8531	0.9476				
-20	132.7	0.0007362	0.1474	25.49	212.92	0.1046	0.8411	0.9457				
-18	144.6	0.0007396	0.1359	28.09	211.55	0.1148	0.8292	0.9440				
-16	157.3	0.0007430	0.1255	30.69	210.18	0.1250	0.8174	0.9423				
-14	170.8	0.0007464	0.1161	33.30	208.79	0.1350	0.8057	0.9407				
-12	185.2	0.0007499	0.1074	35.92	207.39	0.1451	0.7941	0.9392				
-10	200.6	0.0007535	0.0996	38.55	205.97	0.1550	0.7827	0.9377				
-8	216.9	0.0007571	0.0924	41.19	204.53	0.1650	0.7714	0.9364				
-6	234.3	0.0007608	0.0859	43.84	203.08	0.1749	0.7602	0.9351				
-4	252.7	0.0007646	0.0799	46.50	201.61	0.1848	0.7490	0.9338				
-2	272.2	0.0007684	0.0744	49.17	200.12	0.1946	0.7380	0.9326				
0	292.8	0.0007723	0.0693	51.86	198.60	0.2044	0.7271	0.9315				
2	314.6	0.0007763	0.0647	54.55	197.07	0.2142	0.7162	0.9304				
4	337.7	0.0007804	0.0604	57.25	195.53	0.2239	0.7055	0.9294				
6	362.0	0.0007845	0.0564	59.97	193.95	0.2336	0.6948	0.9284				
8	387.6	0.0007887	0.0528	62.69	192.36	0.2432	0.6842	0.9274				
12	443.0	0.0007975	0.0463	68.19	189.11	0.2625	0.6632	0.9256				
16	504.5	0.0008066	0.0408	73.73	185.74	0.2816	0.6424	0.9240				
20	571.7	0.0008161	0.0360	79.32	182.28	0.3006	0.6218	0.9224				

R134a - TetraFluoroEthane Saturation Properties - Temperature Table (20°C - 101.06°C)

Temp °C	Pressure kPa	volume (m <sup>3</sup> /kg)		enthalpy (kJ/kg)					entropy (kJ/kg.K)		
		vf	vg	hf	hfg	hg	sf	sfg	sg		
20	571.7	0.008161	0.0360	79.32	182.26	261.60	0.3006	0.6218	0.9224		
24	645.8	0.008261	0.0319	84.98	178.70	263.68	0.3196	0.6014	0.9210		
26	685.4	0.008313	0.0300	87.83	176.87	264.7	0.3290	0.5912	0.9203		
28	726.9	0.008367	0.0283	90.70	175.00	265.69	0.3388	0.5811	0.9196		
30	770.2	0.008421	0.0266	93.58	173.09	266.67	0.3479	0.5710	0.9189		
32	815.4	0.008478	0.0251	96.48	171.16	267.64	0.3573	0.5609	0.9182		
34	862.6	0.008536	0.0237	99.40	169.18	268.58	0.3667	0.5508	0.9175		
36	911.9	0.008595	0.0224	102.33	167.17	269.50	0.3761	0.5407	0.9168		
38	963.2	0.008657	0.0211	105.29	165.12	270.41	0.3855	0.5307	0.9162		
40	1016.6	0.008720	0.0200	108.27	163.01	271.28	0.3949	0.5206	0.9155		
42	1072.2	0.008786	0.0189	111.26	160.88	272.14	0.4043	0.5105	0.9147		
44	1130.1	0.008854	0.0178	114.28	158.69	272.97	0.4136	0.5004	0.9140		
48	1252.9	0.008997	0.0160	120.39	154.16	274.55	0.4324	0.4800	0.9125		
52	1383.4	0.009150	0.0143	126.60	149.41	276.01	0.4513	0.4595	0.9108		
56	1528.2	0.009317	0.0128	132.92	144.40	277.32	0.4702	0.4387	0.9089		
60	1681.8	0.009498	0.0114	139.36	139.13	278.49	0.4892	0.4176	0.9068		
70	2116.8	0.010038	0.0087	156.14	124.37	280.31	0.5376	0.3624	0.9000		
80	2633.2	0.010775	0.0064	174.25	106.42	280.67	0.5880	0.3014	0.8894		
90	3244.2	0.011936	0.0046	194.78	84.49	277.27	0.6434	0.2272	0.8706		
100	3972.4	0.013557	0.0027	225.15	34.39	259.34	0.7232	0.0921	0.8135		
101.06	4059.1	0.013535	0.0020	241.49	0	241.49	0.7665	0	0.7665		